

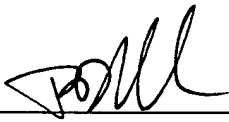
pre-assembled between the hub 3 and the shaft 4. In an internal bore 31, the outer race 51 of the bearing is pressed against a collar 32 of the hub 3, while the inner race 52 having a conical-roller race [53] 54 is slipped on the shaft 4.”

Please amend paragraph 5 of page 5 of the specification as follows:

“The second embodiment of a cartridge-type seal according to the invention shown in fig. 2 differs from the cartridge-type seal according to fig. 1 in two respects. First, a radial sealing lip corresponding to sealing lip 15 of the embodiment shown in fig. 1 is omitted, whereby a source of frictional heat is eliminated, and only two spaces [30, 32] 33, 35 filled with grease are left between the radial sealing lip 14 and the axial protective lip 17, on the one hand, and the protective lip 17 and the axial sealing lip 16, on the other hand. Secondly, instead of a rubber seat 25 for the second casing member 2, a metallic snug fit is provided between the second casing member 2 and the shaft 4, especially in the area beneath the sealing lip 14. That permits direct dissipation of the resulting frictional heat between the sealing lip 14 and the contact surface 22 into the shaft 4, as is not possible with the embodiment according to fig. 1 because the rubber seat 25 is a poor thermal conductor. Yet it is a disadvantage of the embodiment according to fig. 2 that the inner diameter of the axial leg 21 of the casing member 2 and the outer

diameter of the shaft 4 must be manufactured to considerably more restricted tolerances in order to always provide a close fit.”

Respectfully submitted,

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Dated: November 13, 2003